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Journal of Sound and Vibration (1980) 72(4), 547-549

① 2 Jun 80 ⑫ 3

COMMENTS ON "VIBRATION OF A ROTATING BEAM WITH TIP MASS"

Recently Hoa [1] presented a finite element model to obtain the vibration frequencies of a rotating beam with tip mass. It was discovered later that there was a minus sign missing in the expression for d in Table 1 of reference [1][†]; the correct expression should be $d = -(R' + n)$. The correct expression for d gives faster convergence in the finite element results and slightly smaller values than those reported in reference [1]. A revised version of Table 3 of reference [1] is shown in Table 1. In Table 1, a value of $\lambda = 10.70310$ is given instead of the value of $\Omega = 500$ rad/s in Table 3 of reference [1] based upon the physical parameters of $h = 0.02$ in, $L = 5$ in, $E = 30 \times 10^6$ lb/in², $\rho = 0.283$ lb/in³, and $g = 386$ in/s².

TABLE 1

Convergence of the finite element model, $\lambda = 10.70310$, $R = \gamma = 0$

Mode	Vibration frequency parameter $\bar{\alpha}$		
	10 elements	12 elements	14 elements
First	11.8926	11.8924	11.8924
Second	35.0044	35.0037	35.0034
Third	76.3471	76.3394	76.3360
Fourth	136.889	136.835	136.811
Fifth	216.963	216.726	216.621
Sixth	317.204	316.446	316.101

TABLE 2

Comparison between the present finite element results and calculations based on [2], first mode, $R = \gamma = 0$

λ	$\bar{\alpha}$		$M = 1, N = 15$ [2] (exact)
	Present, 10 elements	$M = 10, N = 3$ [2]	
0	3.51602	3.51602	3.51602
1	3.68165	3.68165	3.68165
2	4.13732	4.13732	4.13732
3	4.79728	4.79728	4.79728
4	5.58501	5.58501	5.58500
5	6.44957	6.44957	6.44954
6	7.36041	7.36041	7.36037
7	8.29970	8.29970	8.29964
8	9.25694	9.25694	9.25684
9	10.2258	10.2258	10.2257
10	11.2026	11.2026	11.2023
11	12.1847	12.1847	12.1843
12	13.1706	13.1706	13.1702

[†] The results of reference [1] were first noted to be incorrect by Dr Dewey H. Hodges. The actual sign error was later discovered by Dr Michael J. Rutkowski.

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For the case of a rotating beam without a tip mass ($\gamma = 0$), the present finite element results obtained by using a third order polynomial and ten elements are presented for various values of the angular velocity parameter λ . Results obtained by using Hodges' analysis [2] with both the present element configuration ($M = 10$, $N = 3$) and with the fifteenth order polynomial and one element ($M = 1$, $N = 15$), which yields the essentially exact solution, are also shown. For ten elements, the results obtained from the present analysis and that of reference [2] are identical and agree quite well with the exact solution.

TABLE 3
*Comparison between the present finite element results and calculations
based on [2], second mode, $R = \gamma = 0$*

λ	$\bar{\alpha}$		
	Present, 10 elements	$M = 10$, $N = 3$ [2]	$M = 1$, $N = 15$ [2] (exact)
0	22.0352	22.0352	22.0345
1	22.1817	22.1817	22.1810
2	22.6156	22.6156	22.6149
3	23.3210	23.3210	23.3203
4	24.2740	24.2740	24.2733
5	25.4468	25.4468	25.4461
6	26.8098	26.8098	26.8091
7	28.3349	28.3349	28.3341
8	29.9963	29.9963	29.9954
9	31.7716	31.7716	31.7705
10	33.6416	33.6416	33.6404
11	35.5905	35.5905	35.5890
12	37.6050	37.6050	37.6031

TABLE 4
*Comparison between the present finite element results and calculations
based on [2], first mode, $R = 1$, $\gamma = 0$*

λ	$\bar{\alpha}$		
	Present, 10 elements	$M = 10$, $N = 3$ [2]	$M = 1$, $N = 15$ [2] (exact)
0	3.51602	3.51602	3.51602
1	3.88883	3.88883	3.88882
2	4.83369	4.83369	4.83369
3	6.08177	6.08177	6.08175
4	7.47509	7.47509	7.47505
5	8.94046	8.94046	8.94036
6	10.4441	10.4441	10.4439
7	11.9694	11.9694	11.9691
8	13.5079	13.5079	13.5074
9	15.0549	15.0549	15.0541
10	16.6075	16.6075	16.6064
11	18.1642	18.1642	18.1625
12	19.7237	19.7237	19.7215

The results in Tables 1-4 are for $\bar{\alpha} = (\alpha^2 + \lambda^2 \sin^2 \theta)^{1/2}$ and are thus valid for all θ . The definitions of α , λ , R , and θ are the same as in reference [1].

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(Received 2 June 1980)

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2. D. H. HODGES 1979 *Journal of the American Helicopter Society* **24**, 43-50. Vibration and response of nonuniform rotating beams with discontinuities.

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